



Sikafloor® Marine

SOUND TEST REPORT

Sikafloor® Marine Visco EM 5

VEM X + 3mm Steel

PERFORMED BY DELTA Acoustics



DELTA Test Report



Sound insulation properties of Sikafloor® Marine flooring constructions

Sikafloor® Marine Visco EM 5

Performed for Sika Services AG

DANAK 100/2404 Revision1

Project no.: 118-21873

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23 October 2018

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Sound insulation properties of Sikafloor® Marine flooring constructions
Sikafloor® Marine Visco EM 5

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Summary

The airborne sound insulation and the impact sound insulation are measured according to the ISO and ASTM standards.

Furthermore, the structure-borne sound properties are measured according to ASTM standards and measuring procedure applied by DELTA Acoustics.

Remark

The test results apply only to the objects tested.

This report is a revision and replaces previously issued report dated 19 June 2018.

The change in this report: Graph sheet 2 has been replaced due to incorrect data.

DELTA – a part of FORCE Technology, 23 October 2018



Leif Ødegaard
Acoustics

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1. Introduction

This test report describes the results and procedures for measurements of acoustical and structural vibration properties for marine flooring systems.

2. Test facilities and methods

2.1 Standards

The airborne sound insulation, the impact sound insulation and the structure-borne sound insulation are measured according to the following standards and methods:

- 1) ISO 10140:2015: “Acoustics – Laboratory measurement of sound insulation of building elements” - Part 1,
- 2) ISO 10140:2010: “Acoustics – Laboratory measurement of sound insulation of building elements” -part 2, 4 and 5.
- 3) ISO 717:2013: “Acoustics – Rating of sound insulation in buildings and of building elements” - Part 1 and 2.
- 4) “Procedure for measurement of acoustical and structural properties of maine flooring systems”, DELTA Technical Note, TC-100853.
- 5) ASTM E2963-16: “Standard Test Method for Laboratory Measurement of Acoustical Effectiveness of Ship Noise Treatments – Laboratory Measurement of Acoustical Effectiveness for Marine Bulkhead and Deck Treatments”.
- 6) ASTM E756-5(2010): “Standard Test Method for Measuring Vibration-Damping Properties of Materials”.

2.2 Test facilities

The measurements are carried out in two reverberant rooms at the Technical University of Denmark, 2800 Kgs. Lyngby.

The rooms are built on two separate foundations made of concrete with a wall thickness of 30 cm. Between the source room and the receiving room there is an opening of 2.99 m x 3.37 m, i.e. in the ceiling of the source room and in the floor of the receiving room, see Figure 1.

The volume of the source room and receiving room is 243 m³ and 230 m³, respectively.

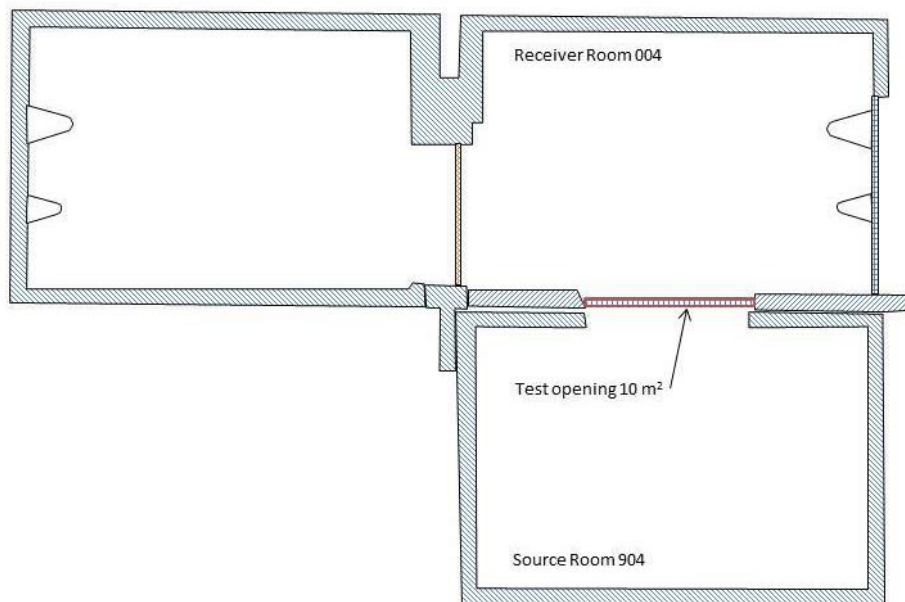


Figure 1

A sketch of the measurement rooms at the Technical University of Denmark.

Excitation of the deck with airborne noise and impact noise is carried out with loudspeakers and a tapping machine as stated in ISO 10140:2010.

Excitation of the deck with structure-borne noise is performed by means of a vibration exciter coupled to a steel plate, which is mounted perpendicularly and below the steel deck positioned in the opening. By means of this arrangement a reverberant vibrational field is established both in the steel plate coupled to the exciter and the steel deck simulating the real conditions occurring in a ship structure. A sketch of the arrangement is shown in Figure 2.

The steel deck is stiffened by 4 flat bars spaced 740 mm in the longitudinal direction. The steel deck is elastically mounted in the test opening. The gap between the opening and steel deck is sealed with mineral wool and tape. The remaining visible part of the supporting steel beams towards receiving room are covered by a heavy rubber matt to avoid flanking noise radiation.

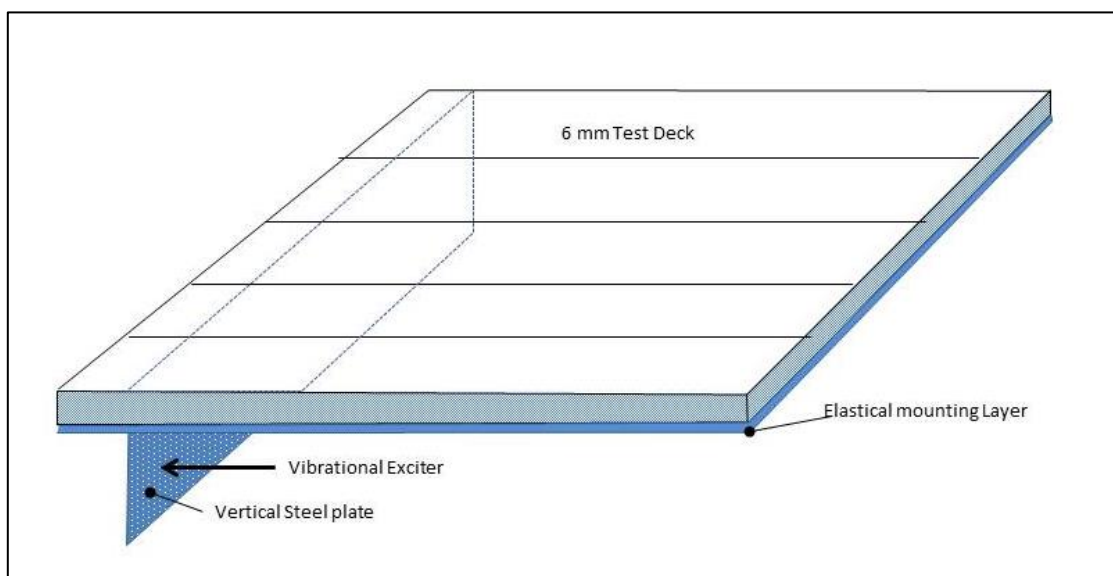


Figure 2
A sketch of the arrangement consisting of the 6 mm test deck, the elastic mounting of the deck and the 6 mm steel plate coupled to the test deck. The arrow indicates the position of the electro-dynamic exciter.

2.3 Measurement methods

2.3.1 ISO 10140:2010

During the airborne and structure-borne sound measurements the excitation is performed by means of broadband pink noise in the frequency range 20-10000 Hz.

The response, i.e. the sound pressure level in the receiving room for the airborne and impact sound insulation measurements or the velocity level on the floor for the structure-borne sound measurements, will be measured in one-third octave filter bands with centre frequencies from 50 Hz to 5000 Hz.

Measurements in the one-third octave filter bands of 50 Hz, 63 Hz and 80 Hz are not required according to ISO 10140:2010. However, based on experience from previous measurements on ships, it seems reasonable to include these frequency ranges.

Due to the volume of the test rooms, some additional uncertainty occurs for the measurements in the one-third octave filter bands of 50 Hz, 63 Hz and 80 Hz. In Section 2 the measurement results from these frequency bands are therefore presented for information only.

All relevant instruments in the test setup are calibrated before and during the testing period for every construction.

2.3.1.1 Airborne sound insulation

The airborne sound insulation is normally specified by the sound reduction index, R , as defined according to the ISO 10140 series:

$$R = L_1 - L_2 + 10 \log(S/A) \text{ dB}$$

where

L_1 = average sound pressure level in the source room

L_2 = average sound pressure level in the receiving room

S = area of the test floor, which was 10 m^2

A = equivalent absorption area in m^2 in the receiving room

From the measured values of R , the weighted sound reduction index, R_w (formerly: airborne sound insulation index I_a) is calculated. The calculations follow the procedure as stated in ISO 717-1:2013.

2.3.1.2 Impact sound insulation

The normalized impact sound pressure level, L_n , is defined as the impact sound pressure level, L_i , increased by a correction term given in decibels and being ten times the common logarithm of the ratio between the measured equivalent absorption area A of the receiving room and the reference equivalent absorption area A_0 , i.e.

$$L_n = L_i + 10 \log(A/A_0) \text{ dB} \quad (1)$$

where

$$A_0 = 10 \text{ m}^2$$

For each measurement series the weighted normalised impact sound pressure level $L_{n,w}$ (formerly: impact sound index I_i) is calculated as stated in ISO 717-2:2013.

2.4 Measurement of structure-borne sound properties

No international ISO standard exists for measurement of structure-borne sound insulation properties for marine floors and bulkheads. Consequently, this test will be carried out by means of a method previously performed by DELTA and used for similar constructions.

Vibrational power is supplied to the steel deck by means of the arrangement described in Figure 2. The supply of constant vibrational power is monitored during the measurement period by means of a force transducer mounted between the vertical steel plate and the vibration exciter. Further, the acceleration level at the input position on the vertical steel plate is measured for monitoring purpose.

The response is measured as the velocity level, L_v in dB re 10^{-9} m/s in minimum 24 different positions on the test surface, on the steel deck below the test construction and on the floor.

Measurement positions are selected pseudo-randomly on the test structure.

2.4.1 Effect of treatment

From the average value of the velocity level measured on the test surface the transmission loss TL_v and insertion loss IL_v and IL_p will be calculated. Further, the radiation efficiency will be calculated.

The measured transmission loss TL_v in dB for the constructions describes the difference between the velocity level on the steel deck after installation of the floor construction and the velocity level measured on top of the floor covering. Thus, the transmission loss expresses the reduction in the velocity level from the steel deck to the floor covering

The transmission loss TL_v for a structure during test is calculated using the following formula:

$$TL_v = L_{v,above} - L_{v,below}$$

where

$L_{v,above}$ = time and space average vibration velocity on top of the test construction in the receiver room, dB re: 1 nm/s

$L_{v,below}$ = time and space average vibration velocity below of the test construction in the source room, dB re: 1 nm/s.

The measured insertion loss IL_v in dB describes the difference between the velocity level measured on the bare steel deck before installation of the floor construction and the velocity level measured on top of the applied floor construction. The insertion loss IL_v describes the improvement of the vibration level on the floor achieved by using the floor covering.

The insertion loss, IL_v , for a structure during test is calculated using the following formula:

$$IL_v = L_{v,above} - L_{v,ref}$$

where

$L_{v,above}$ = time and space average vibration velocity on top of the test construction in the receiver room, dB re: 1 nm/s

$L_{v,ref}$ = time and space average vibration velocity of the bare steel deck before application of the test construction room, dB re: 1 nm/s.

The measured insertion loss, IL_p , in dB regarding radiated structure-borne sound to the room describes the difference between the measured radiated sound pressure level in the receiving room before installation of the floor covering and the measured radiated sound pressure level after applying the floor covering. The insertion loss, IL_p , thus expresses the improvement of the sound level in the room above the deck achieved by using the floor covering.

The insertion loss, IL_p , for a structure-borne radiated noise is calculated using the following formula:

$$IL_p = L_{p,test\ construction} - L_{p,ref}$$

where

$L_{p,test\ construction}$ = averaged sound pressure level in dB re 20 μ Pa in the receiving room with the test construction mounted on the steel reference deck.

$L_{p,ref}$ = averaged sound pressure level in dB re 20 μ Pa in the receiving room with the bare steel deck without the test construction mounted.

2.4.2 Radiation efficiency

The radiation index describes the ability of a vibrating floor to radiate sound. A high radiation index combined with a high velocity level on the floor covering causes high noise levels in the rooms above the deck covering.

The radiation efficiency is normally expressed as a logarithmic quantity named the radiation index, $10\log\sigma$. If the radiation index is determined from sound power measurements in a reverberant room, this can be calculated using the following formula:

$$10\log\sigma = L_w - L_v - 10\log(S/1\text{m}^2) + 34 \text{ dB}$$

or based on the averaged sound pressure level in the receiving room:

$$10\log\sigma = L_p - L_v + 10\log V - 10\log T - 10\log(S/1\text{m}^2) + 10 \log(1 + F \lambda/8V) + 20 \text{ dB}$$

where

L_w = averaged sound power in dB re 1pW

L_p = averaged sound pressure level in dB re. $20\mu\text{Pa}$ in the receiving room

L_v = averaged velocity level in dB re 1mm/s measured on the surface of the covering floor

S = area of the test floor, which is 10 m^2

V = volume in m^3 of the receiving room, which is 230 m^3

T = reverberation time in seconds

F = total area in m^2 of the surface in the receiving room, which is 300 m^2

λ = wavelength in m of the centre frequency of the one-third octave filter band in question.

$10\log(1 + F \lambda/8V)$ is normally called the Waterhouse correction

Due to a very high damping in some of the tested constructions, the radiated structure-borne noise from the floor coverings can be influenced by flanking noise contribution from the test rooms. This phenomenon takes place in the high frequency range above 2 kHz.

Consequently, the radiated sound pressure level in the receiving room can optionally be determined using intensity measuring technique. The radiation index might in these situations be calculated based on the measured sound power level and not on basis of the measured sound pressure level.

2.5 ASTM E2963-16

The full scale test of marine flooring constructions are expensive and are normally done for a number of different flooring constructions mounted successive upon the reference steel deck. This allows comparing the different flooring constructions directly. However, this means that the reference deck must not be removed from the test opening during the measuring series. This is necessary in order not to introduce differences due to the mounting in the test facilities.

Measurements of transmission loss and acceptance will be performed simultaneously with airborne noise excitation in the source room. Measurements of sound absorption are done in connection with the transmission loss measurements. Measurements of transmission loss are performed in accordance with ASTM E90-09.

Primarily the damping properties for the constrained damped test constructions will be determination of the loss factor using the test beam method e.g. as described in ASTM E756-5(2010).

The loss factor cannot be evaluated for floating floors, as the loss factor does not describe the vibration damping properties for such floor systems.

All calculations are performed for each one-third octave band frequency.

2.5.1 Transmission loss

According to ASTM E90-09 the transmission loss for a structure during test is calculated using following formula:

$$TL = L_1 - L_2 + 10 \log[S/A_2]$$

where

TL = transmission loss of the structure, dB

L₁ = time and space average sound pressure level in the source room,
dB re 20 μPa

L₂ = time and space average sound pressure level in the receiver room,
dB re 20 μPa

S = surface area of the test structure, m²

A₂ = equivalent absorption area in m² in the receiving room

2.5.2 Acceptance

Measurements of acceptance are performed by generating an acoustic signal in the source room and measuring the generated sound pressure level in the source room as well as the surface vibration of the test structure.

The acceptance of a structure during test is defined here as assuming a reverberant receiver room:

$$L_{\Lambda} = L_1 - L_v$$

where

L_{Λ} = acceptance of the structure, dB re 20 μ Pa/10nm/s

L_1 = time and space average sound pressure level in the source room, dB re 20 μ Pa, and

L_v = time and space average surface vibration velocity level on the test structure, dB re 10 nm/s.

For each measurement of vibration, the measured acceleration level will be converted to velocity using the equation:

$$L_v = L_a - 20 * \text{Log}(2 * \pi * f) + 60$$

where

L_v = vibration velocity level in dB re 10 nm/s

L_a = vibration acceleration level in dB re 10 μ m/s²

f = one-third-octave band centre frequency.

The space and time averaged vibration velocity level will be calculated in each one-third-octave band.

2.5.3 Radiation efficiency

Measurements of radiation efficiency will be performed separately with structure-borne noise excitation with the exciter system as described in Section 2.2.

Measurements of radiation efficiency are performed by energizing the vibration exciter and measuring the responding vibration of the test structure as well as the sound pressure level in the receiving room.

The calculation of radiation efficiency for a structure under test uses the equation:

$$L_{\sigma} = L_2 - L_v - 10 * \text{Log}[4 * S / A_2] + 13.7$$

where

L_{σ} = radiation efficiency of the structure, dB re 20 μ Pa/10 nm/s

L_2 = time and space average sound pressure level in the receiver room, dB re 20 μ Pa

L_v = time and space average vibration velocity level in the receiver room, dB re 10 nm/s

S = surface area of the test structure, m^2

A_2 = equivalent absorption area in m^2 in the receiving room.

The same values as stated in Section 2.4.2 will be obtained except for the Waterhouse correction used in Section 2.4.2.

2.5.4 Absorption

The change in absorption will be evaluated based on the measurement of the reverberation time in the receiving room (treated side of construction) and the calculated absorption area. An absorption coefficient α will be calculated based on the reverberation time with the bare steel deck installed and the reverberation time with the floor construction applied.

The calculation of treatment absorption is performed using the following equation:

$$\alpha = (A_{\text{Treat}} - A_{\text{No treat}}) / S_{\text{Treat}}$$

where

A_{Treat} = equivalent sound absorption area after the treatment has been applied (m^2)

A_{NoTreat} = equivalent sound absorption area in the same room prior to application of the treatment (m^2)

S_{Treat} = surface area of the test structure (m^2).

2.5.5 Damping for constrained damped constructions

Primarily, the damping properties for the constrained damped test constructions will be determined as the loss factor using the test beam method e.g. as described in ASTM E756-5(2010). Based on the measured values a regression analysis is performed in order to get estimated one-third octave values for the loss factor.

Alternatively, the damping properties of the constrained damped test constructions will also be evaluated by measuring the total loss factor using the guide lines described in ISO 10848-1:2006: “Acoustics -- Laboratory measurement of the flanking transmission of airborne and impact sound between adjoining rooms -- Part 1: Frame document - Section 7.3 Measurement of structural reverberation time”.

The excitation method is vibrator excitation with the exciter mounted as described in Section 2. The impulse response is measured e.g. with the MLS (Maximum Length Sequence) or vibration sweep techniques. The integrated impulse response method is used with backward integration of the squared impulse response as defined in ISO 3382: “Measurement of the reverberation time of rooms with reference to other acoustical parameters”.

The relation between the total loss factor and the structural reverberation time is as follows:

$$\eta_{\text{total}} = 2.2 / (f * T_s)$$

where

f = frequency in Hz

T_s = structural reverberation time in seconds.

This method allows a practical approach for applying damping properties for the test construction without removing the construction from the test opening.

Based on the measured values a regression analysis is performed in order to get estimated one-third octave values for the loss factor.

2.5.6 Damping for floating floor constructions

The loss factor for floating floor constructions cannot be evaluated, as the loss factor does not describe the vibration damping properties for such systems.

An estimate for the structure-borne noise reduction for floating floor constructions might be estimated based on the insertion loss, IL_v , determined as described in section 2.4.1. This is not a part of ASTM E2963-15.

2.5.7 Effect of treatment

Transmission Loss

This equation is used to calculate the effect of the treatment on transmission loss:

$$\Delta TL = TL_{\text{Treat}} - TL_{\text{Non-Treat}}$$

where

ΔTL = change in transmission loss due to the application of the treatment, dB

TL_{Treat} = transmission loss calculated for the test structure with the treatment, dB

$TL_{\text{Non-Treat}}$ = transmission loss calculated for the test structure without the treatment, dB.

Acceptance

This equation is used to calculate the effect of the treatment on acceptance:

$$\Delta L_{\Lambda} = L_{\Lambda, \text{Treat}} - L_{\Lambda, \text{Non-Treat}}$$

where

ΔL_{Λ} = change in acceptance due to the application of the treatment, dB

$L_{\Lambda, \text{Treat}}$ = acceptance calculated for the test structure with the treatment, dB

$L_{\Lambda, \text{Non-Treat}}$ = acceptance calculated for the test structure without the treatment, dB.

Radiation efficiency

This equation is used to calculate the effect of the treatment on radiation efficiency:

$$\Delta L_{\sigma} = L_{\sigma, \text{Non-Treat}} - L_{\sigma, \text{Treat}}$$

where

ΔL_{σ} = change in radiation efficiency due to the application of the treatment, dB

$L_{\sigma, \text{Treat}}$ = radiation efficiency calculated for the test structure with the treatment, dB

$L_{\sigma, \text{Non-Treat}}$ = radiation efficiency calculated for the test structure without the treatment, dB.

Loss factor

The change in damping loss for constrained damped constructions will be computed using the equation:

$$\Delta \eta = \eta_{\text{treated}} - \eta_{\text{non-treated}}$$

where

η = damping loss factor. This will be a function of frequency.

3. Measurement uncertainty

According to EN ISO 12999-1:2014 precision of laboratory measurements expressed as the reproducibility standard deviations are as follows (two-sided 95 % confidence level and $k = 1.96$)

| Value | σ_{R95} ($k = 1.96$, two-sided) |
|-----------|--|
| R_w | ± 2.4 dB |
| $L_{n,w}$ | ± 3.0 dB |

The standard deviations of the structural and acceptance vibration measurements are in the range from 2.5 dB to 4.5 dB depending on frequency and position on the test deck.

The standard deviations of the structural sound measurements are in the range from 0.1 dB to 2.5 dB depending on frequency.

4. General measurement results

4.1 Measurements according to ISO standards and structure-borne sound measuring method

Results of the measurements of the airborne sound insulation, the impact sound insulation and the structure-borne sound insulation for the investigated floor constructions are for each tested construction.

The results are expressed as one-third octave values in the frequency ranges 50 Hz to 5000 Hz and, when appropriate, as a single-number quantity calculated according to the ISO standards.

The results of the sound pressure measurement in the frequency range below 100 Hz are given for the information only, as the dimensions of the two reverberant rooms are too small for measuring precisely in this frequency range.

Furthermore, the results of the measurements on the bare steel deck are indicated with black on each diagram for the airborne sound and the impact sound measurements. The difference between the curves for the test deck and the curves for the reference steel deck thus indicates the improvement in the sound reduction and the impact sound insulation caused by the applied floor construction.

4.2 Measurements according to ASTM E2963-16

The results of the measurements of the airborne sound insulation, acceptance and the radiation efficiency for the investigated floor constructions are for each tested construction.

The results are expressed as one-third octave values in the frequency ranges 50 Hz to 5000 Hz and, when appropriate, as a single-number quantity calculated according to the ISO standards.

The results of the sound pressure measurement in the frequency range below 100 Hz are given for the information only, as the dimensions of the two reverberant rooms are too small for measuring precisely in this frequency range.

The acoustical effectiveness regarding transmission loss, acceptance, radiation efficiency, and absorption and loss factor (where relevant) is calculated based on reference measurements on a bare steel deck for each deck construction.

5. Results for Sikafloor® Marine Visco EM 5

The floor is a constrained damped construction. The constrained damped construction is consisting of approximately 2 mm viscoelastic damping layer type VEM X. The density of compound is 1,300 kg/m³. On top of the damping layer 3 mm steel plates with coverage of 90 % is applied.

The total surface mass is approximately 23.8 kg/m² for the total construction. The total building height is 5 mm.

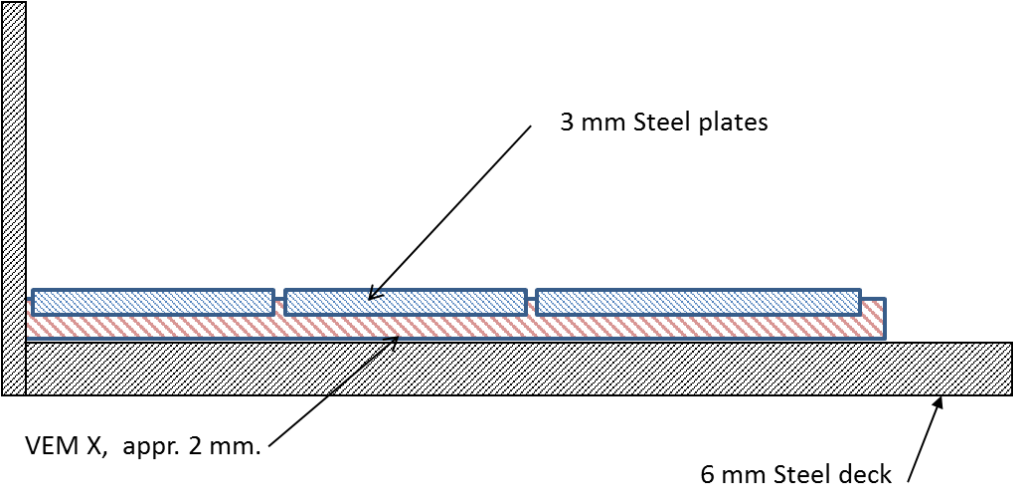


Figure 3
Principle sketch of the test construction.

| Layer | Density kg/m ³ | Thickness mm |
|------------------------------------|---------------------------|--------------|
| Viscoelastic layer VEM X | 1,300 | 2 |
| Constrained layer 3mm steel plates | 7860 | 3 |

Table 1
Product data for the tested construction.

The main results are given in the Graph sheets.

5.1 Results according to ISO standards

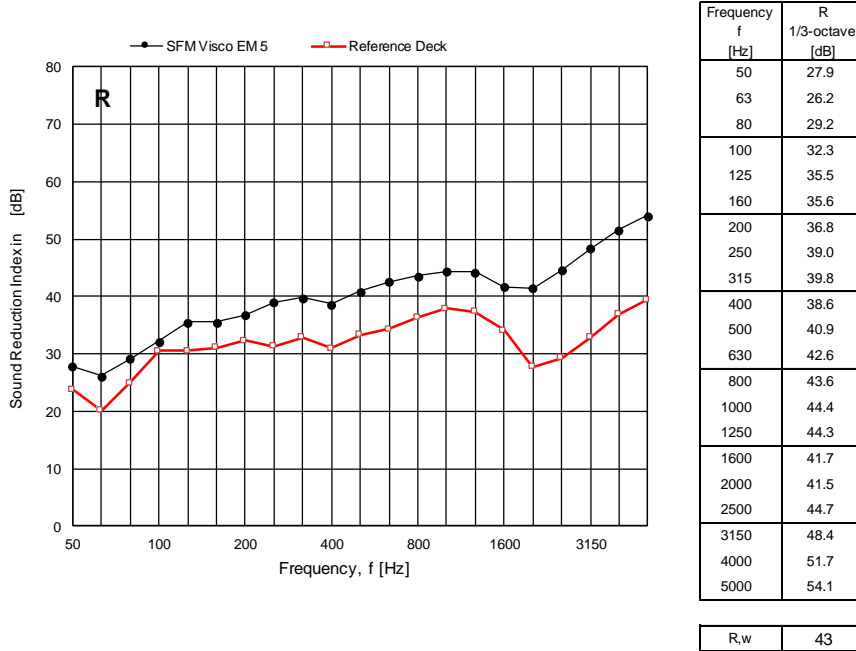


Figure 4
Measured sound reduction index R for the Sikafloor® Marine Visco EM 5 expressed in dB per one-third octave frequency bands. For comparison the results of the measurements on the bare steel deck are also shown.

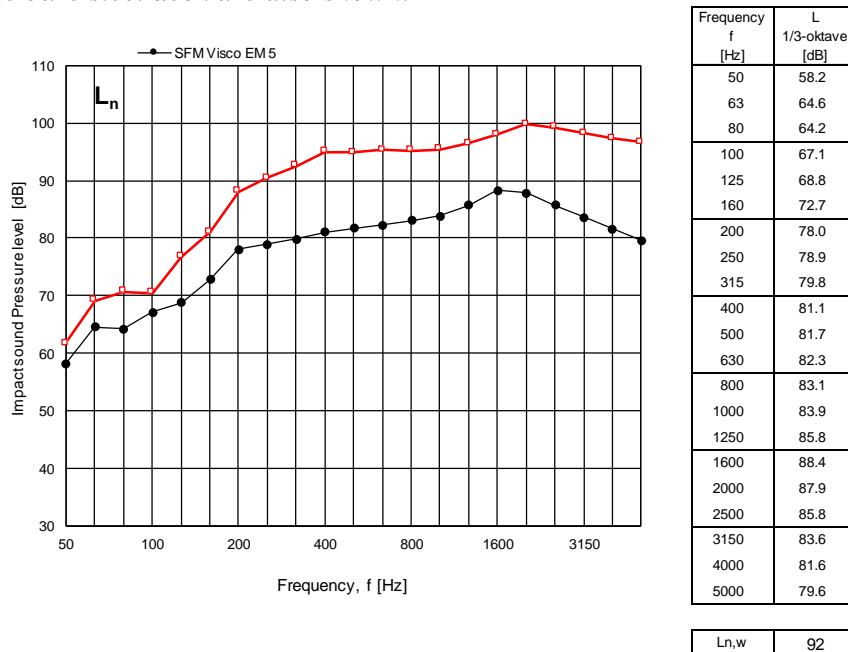


Figure 5
Measured normalized impact sound pressure level L_n for Sikafloor® Marine Visco EM 5 expressed in dB re 20 μ Pa per one-third octave frequency band. For comparison the results of the measurements on the bare steel deck are also shown.

5.2 Structure-borne noise properties

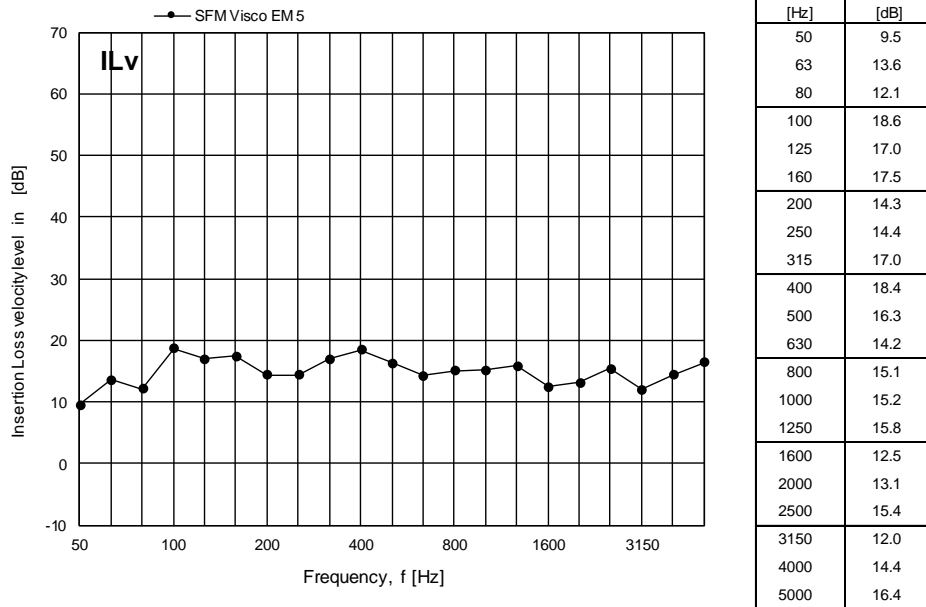


Figure 6
Measured Insertion Loss IL_v for Sikafloor® Marine Visco EM 5 expressed in dB per one-third octave frequency band. The insertion loss IL_v refers to the mean velocity level in dB re 10⁻⁹ m/s on top of the floor covering.

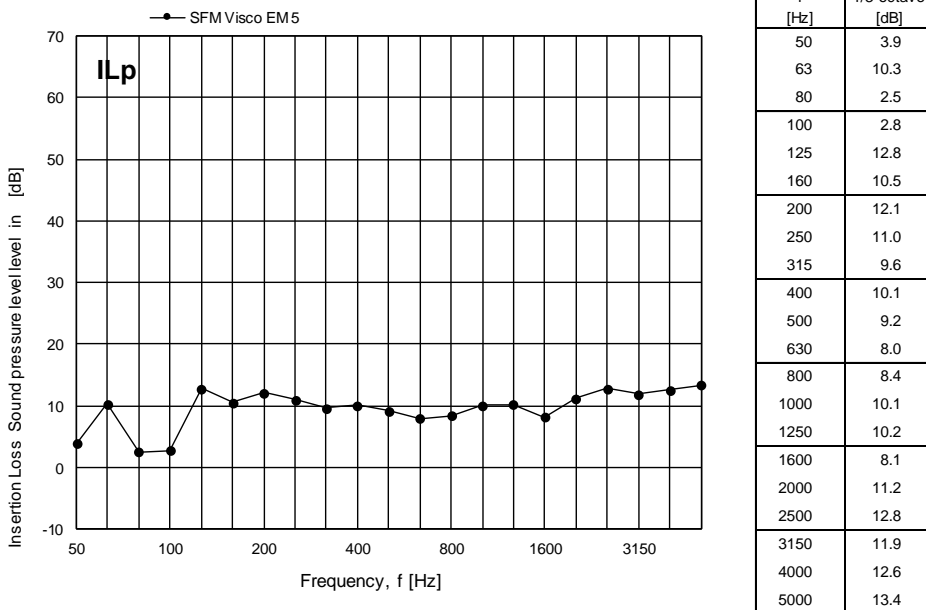


Figure 7
Measured insertion loss IL_p for Sikafloor® Marine Visco EM 5 expressed in dB per one-third octave frequency band. The insertion loss IL_p refers to the radiated mean sound pressure level in dB re 20 μ Pa in the receiving room above the floor.

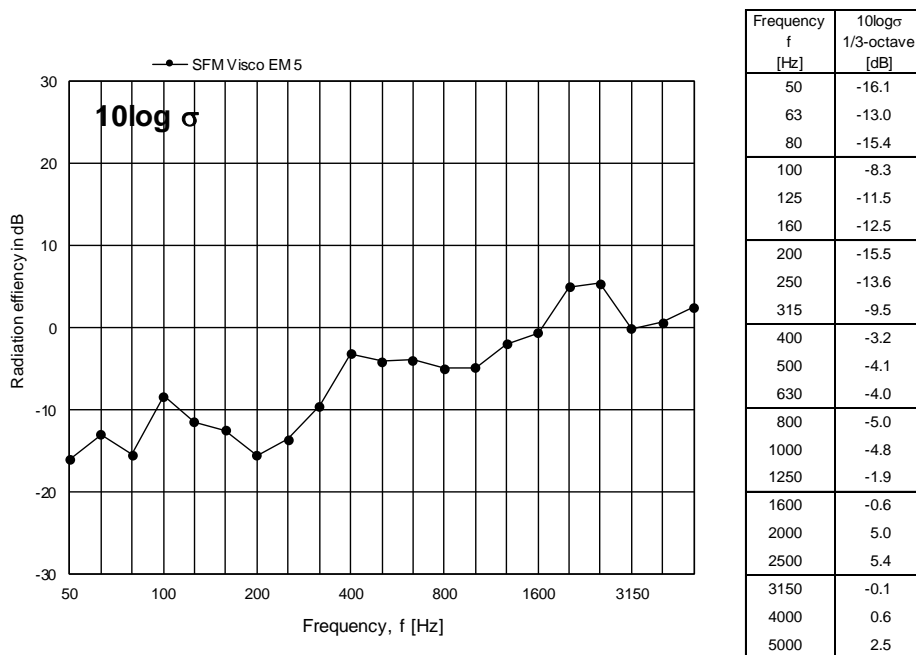


Figure 8
 Measured radiation index $10 \log \sigma$ for Sikafloor® Marine Visco EM 5 expressed in dB per one-third octave frequency band.

5.3 Results according to ASTM E2963-16

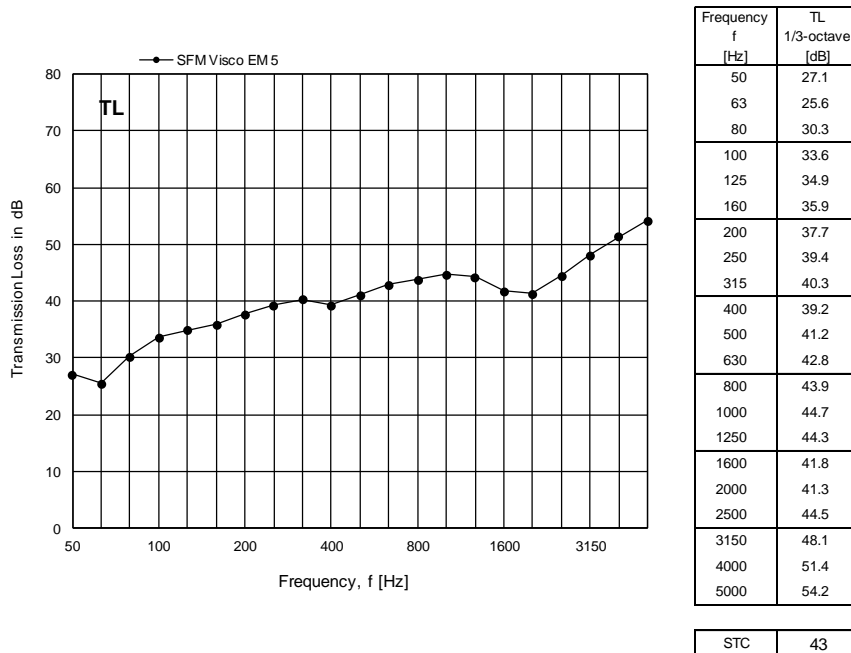


Figure 9
 Measured transmission loss TL for Sikafloor® Marine Visco EM 5 expressed in dB per one-third octave frequency band.

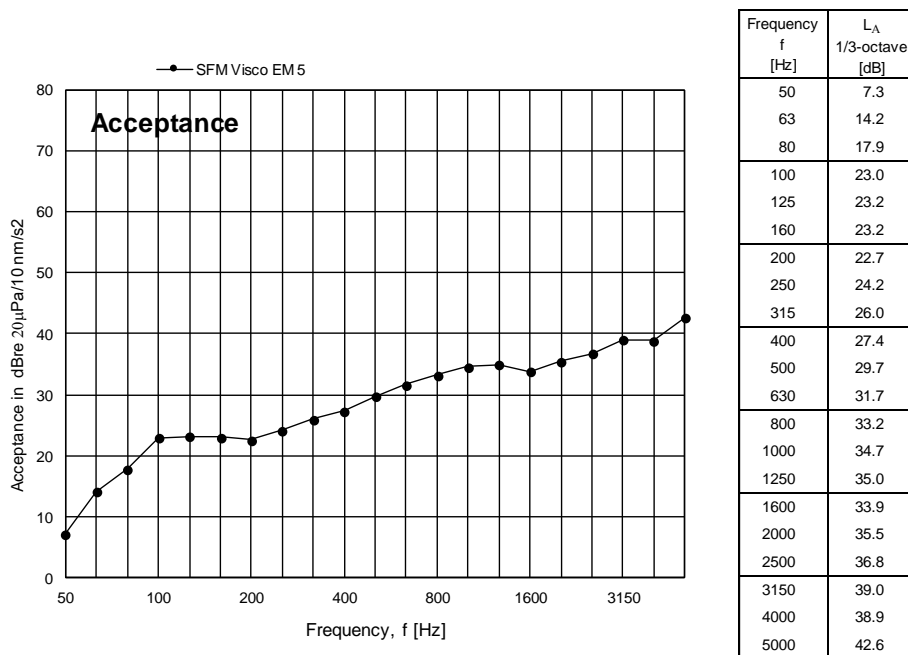


Figure 10
 Measured acceptance L_A for Sikafloor® Marine Visco EM 5 expressed in dB per one-third octave frequency band.

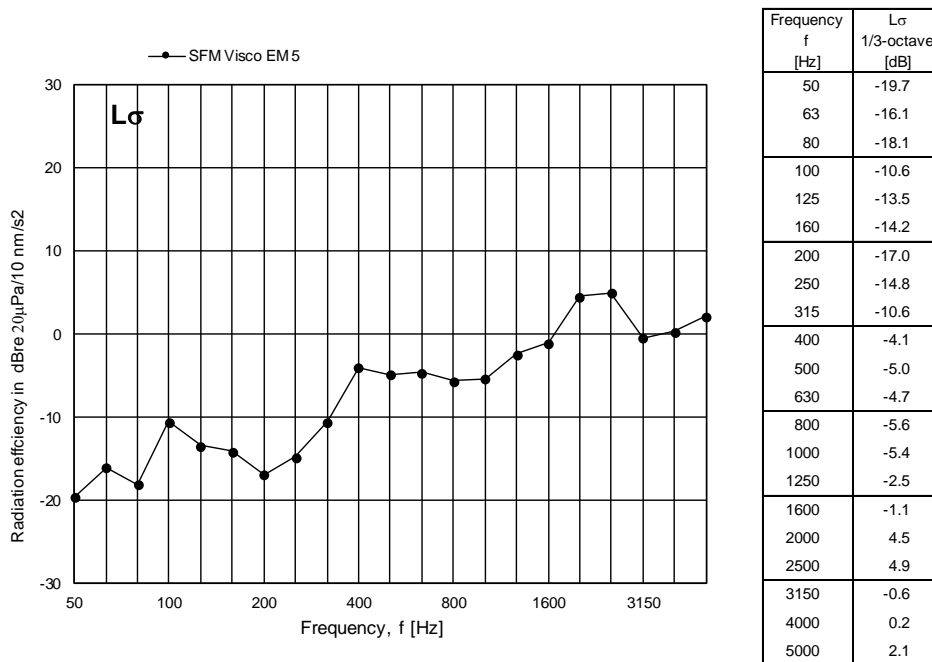
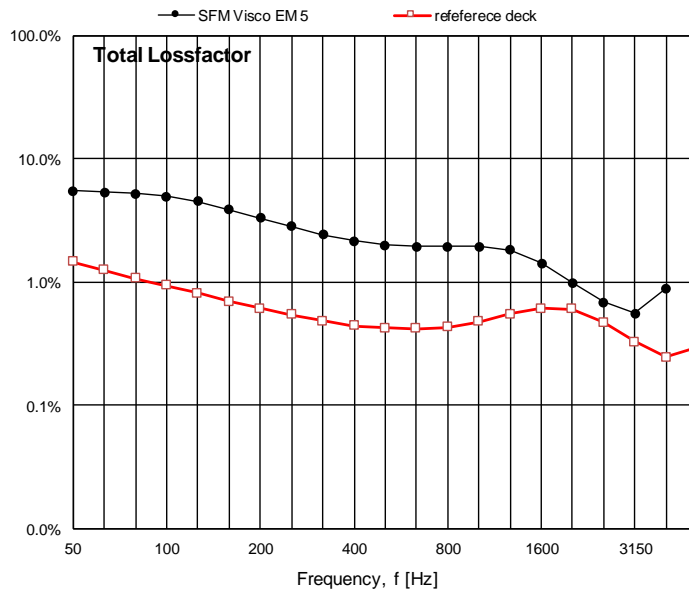


Figure 11
 Measured radiation index L_σ for Sikafloor® Marine Visco EM 5 expressed in dB per one-third octave frequency band.

5.3.1 Loss factor measurements

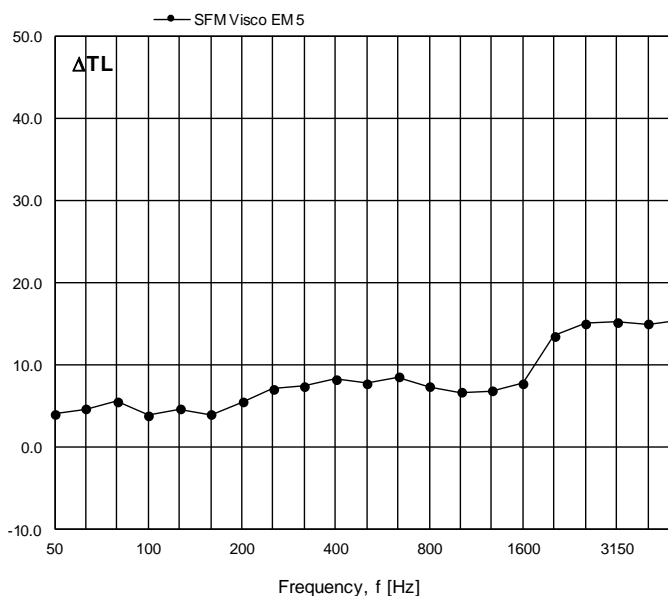


| Frequency f [Hz] | Total Loss factor |
|------------------|-------------------|
| 50 | 5.5% |
| 63 | 5.4% |
| 80 | 5.2% |
| 100 | 5.0% |
| 125 | 4.5% |
| 160 | 3.9% |
| 200 | 3.3% |
| 250 | 2.8% |
| 315 | 2.4% |
| 400 | 2.2% |
| 500 | 2.0% |
| 630 | 1.9% |
| 800 | 1.9% |
| 1000 | 2.0% |
| 1250 | 1.8% |
| 1600 | 1.4% |
| 2000 | 1.0% |
| 2500 | 0.7% |
| 3150 | 0.6% |
| 4000 | 0.9% |
| 5000 | |

Figure 12

Measured total loss factor for test deck applied with constrained damped construction type Sikafloor® Marine Visco EM 5 in % per one-third octave frequency band. For comparison the results of the measurements on the bare 6 mm steel deck are also shown.

5.3.2 Effectiveness



| Frequency f [Hz] | ΔTL dB |
|------------------|--------|
| 50 | 4.0 |
| 63 | 4.7 |
| 80 | 5.5 |
| 100 | 3.8 |
| 125 | 4.6 |
| 160 | 3.9 |
| 200 | 5.5 |
| 250 | 7.1 |
| 315 | 7.5 |
| 400 | 8.3 |
| 500 | 7.8 |
| 630 | 8.6 |
| 800 | 7.3 |
| 1000 | 6.7 |
| 1250 | 6.8 |
| 1600 | 7.8 |
| 2000 | 13.6 |
| 2500 | 15.1 |
| 3150 | 15.3 |
| 4000 | 14.9 |
| 5000 | 15.5 |

Figure 13

Measured changes in transmission loss TL for Sikafloor® Marine Visco EM 5 expressed in dB per one-third octave frequency band.

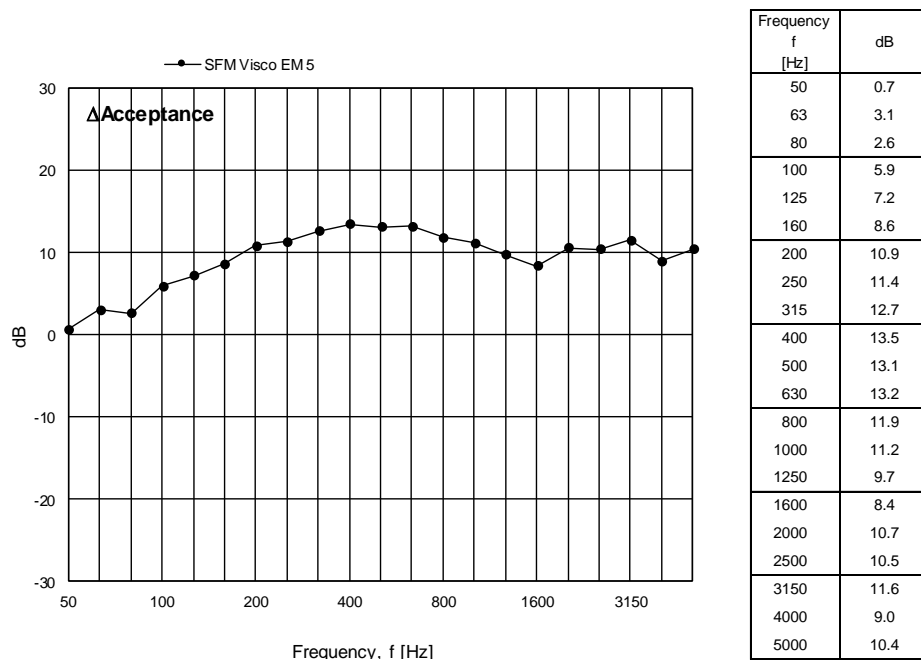


Figure 14
 Measured change in acceptance for SikaFloor® Marine Visco EM 5 expressed in dB per one-third octave frequency band.

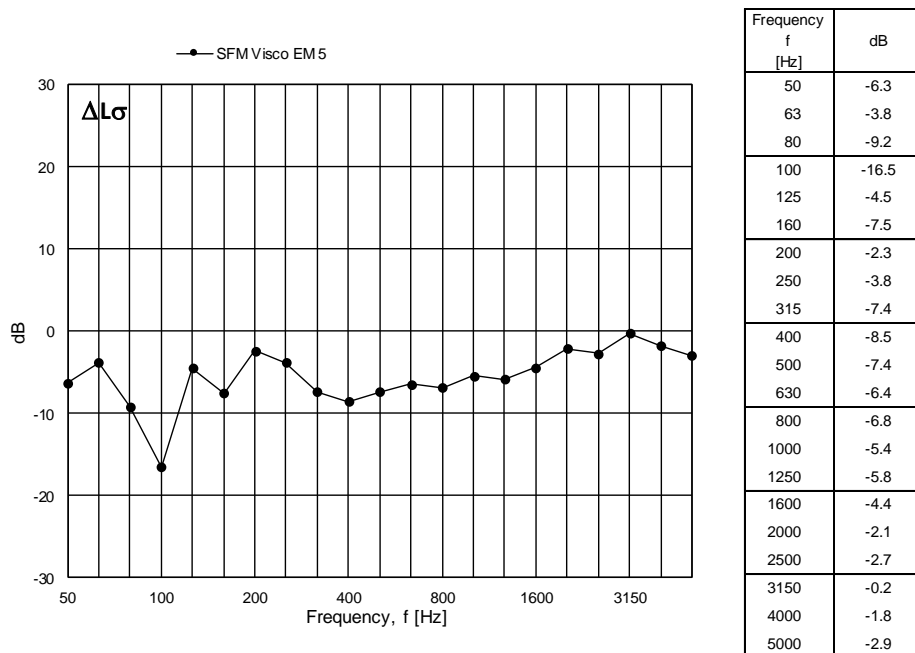
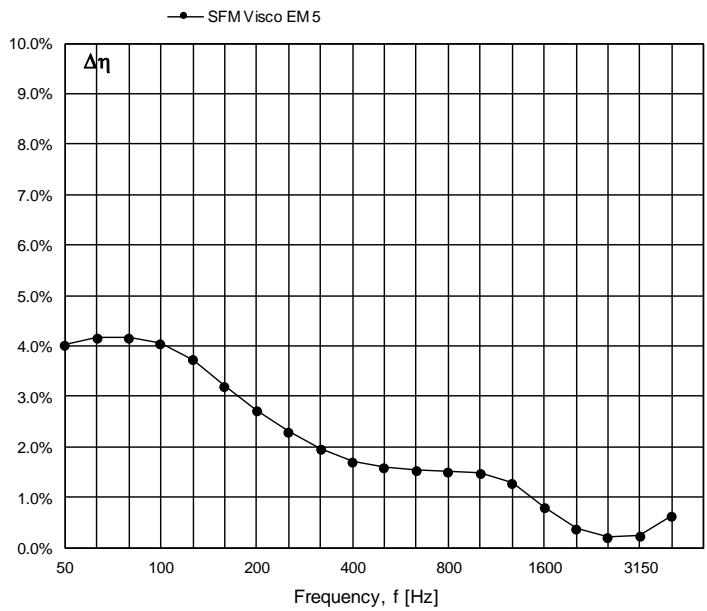
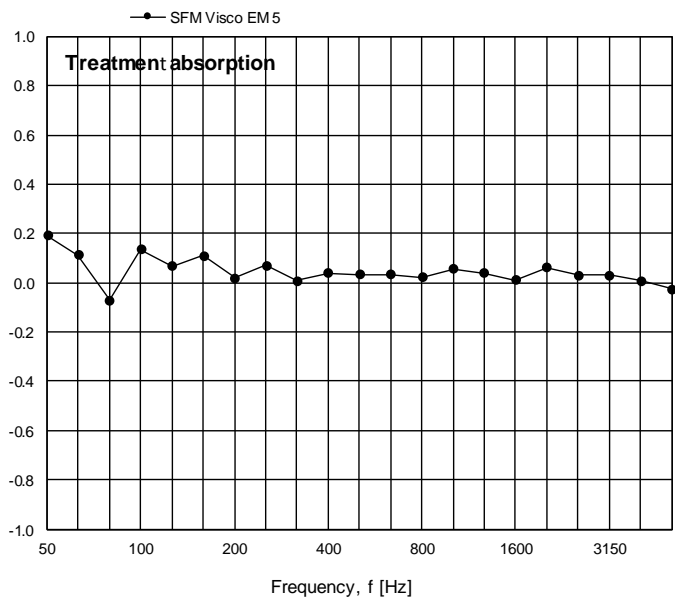


Figure 15
 Measured change in radiation efficiency for SikaFloor® Marine Visco EM 5 expressed in dB per one-third octave frequency band.



| Frequency f [Hz] | Δη |
|------------------|------|
| 50 | 4.0% |
| 63 | 4.2% |
| 80 | 4.2% |
| 100 | 4.0% |
| 125 | 3.7% |
| 160 | 3.2% |
| 200 | 2.7% |
| 250 | 2.3% |
| 315 | 2.0% |
| 400 | 1.7% |
| 500 | 1.6% |
| 630 | 1.5% |
| 800 | 1.5% |
| 1000 | 1.5% |
| 1250 | 1.3% |
| 1600 | 0.8% |
| 2000 | 0.4% |
| 2500 | 0.2% |
| 3150 | 0.2% |
| 4000 | 0.6% |
| 5000 | 0.8% |

Figure 16
Measured change in total loss factor in % for Sikafloor® Marine Visco EM 5 expressed in dB per one-third octave frequency band.



| Frequency f [Hz] | α |
|------------------|-------|
| 50 | 0.19 |
| 63 | 0.11 |
| 80 | -0.07 |
| 100 | 0.13 |
| 125 | 0.07 |
| 160 | 0.11 |
| 200 | 0.02 |
| 250 | 0.07 |
| 315 | 0.01 |
| 400 | 0.04 |
| 500 | 0.03 |
| 630 | 0.03 |
| 800 | 0.02 |
| 1000 | 0.06 |
| 1250 | 0.04 |
| 1600 | 0.01 |
| 2000 | 0.06 |
| 2500 | 0.03 |
| 3150 | 0.03 |
| 4000 | 0.01 |
| 5000 | -0.03 |

Figure 17
Measured treatment absorption for Sikafloor® Marine Visco EM 5 per one-third octave frequency band.

6. Comments

The measured insertion loss for vibration and sound pressure will in practice be better than measured in the laboratory due to the limited size of test deck. The results are in some degree influenced by the short distance to the excitation area of the deck.

7. References

- [1] ISO 10140:2015: “Acoustics – Laboratory measurement of sound insulation of building elements” -- Part 1.
- [2] ISO 10140:2010: “Acoustics – Laboratory measurement of sound insulation of building elements” -- Part 2, 3 and 5.
- [3] ISO 717:2013: “Acoustics – Rating of sound insulation in buildings and of building elements” -- Part 1 and 2.
- [4] “Procedure for measurement of acoustical and structural properties of marine flooring systems”, DELTA Technical Note, TC-100853.
- [5] ASTM E2963-16: “Standard Test Method for Laboratory Measurement of Acoustical Effectiveness of Ship Noise Treatments – Laboratory Measurement of Acoustical Effectiveness for Marine Bulkhead and Deck Treatments”.
- [6] ASTM E756-5(2010): “Standard Test Method for Measuring Vibration-Damping Properties of Materials”.
- [7] ASTM E90-09: “Standard Test Method for Laboratory Measurement of Airborne Sound Transmission Loss of Building Partitions and Elements”.
- [8] ASTM E413-10: “Classification for Rating Sound Insulation”.
- [9] ISO 10848-1:2006: “Acoustics -- Laboratory measurement of the flanking transmission of airborne and impact sound between adjoining rooms” -- Part 1: Frame document.
- [10] ISO 3382: “Measurement of the reverberation time of rooms with reference to other acoustical parameters”.

8. Instrumentation

| Instrument | Type | DELTA No. | Calibration | | |
|---|-------------|-----------|-------------|---------------|------------|
| | | | Latest | Inter-mediate | Next |
| Real-Time Frequency Analyser | B&K 2270 | 1498L | July 2017 | - | July 2019 |
| Measuring Microphone | Gras 40 EN | 1616L | Sep. 2016 | - | Sep. 2018 |
| Measuring Microphone | B&K 4165 | 006S | Sep. 2017 | - | May 2016 |
| Microphone Preamplifier | B&K 2619 | 0707 | Sep. 2017 | - | Sep. 2018 |
| Microphone Preamplifier | B&K 2619 | 815L | Jan. 2018 | - | Jan. 2020 |
| Microphone Power Supply | B&K 5935 | 1040L | Dec. 2016 | - | Dec. 2018 |
| Microphone Power Supply | B&K 5935 | 1392L | Dec 2016 | - | Dec.2018 |
| Sound Level Calibrator | B&K 4231 | 1158L | April 2018 | - | April 2019 |
| Sensor for Temperature and Humidity | EBI-20- TH1 | 1216L | Febr. 2018 | - | - |
| Exiter system | Ling 406 | - | - | - | - |
| Force transducer (monitoring only) | B&K 8200 | 1149733 | - | - | - |
| Accelerometer | IMI 608/A11 | 1501L | March 2016 | - | March 2020 |
| Accelerometer | IMI 608/A11 | 1502L | March 2016 | - | March 2020 |
| Accelerometer | IMI 608/A11 | 1503L | March 2016 | - | March 2020 |
| Accelerometer | IMI 608/A11 | 1504L | March 2016 | - | March 2020 |
| Accelerometer | IMI 608/A11 | 1613L | - | - | - |
| Accelerometer | IMI 608/A11 | 1614L | - | - | - |
| Charge Amplifier | B&K 2635 | 0496T | Jan. 2016 | Jan. 2016 | Jan. 2019 |
| Accelerometer | B&K 4381 | 1587L | March 2015 | - | March 2019 |
| Accelerometer | B&K 4381 | 1588L | March 2015 | - | March 2019 |
| Vibration Calibrator | B&K 4294 | 1414L | Oct. 2015 | Oct. 2015 | Oct. 2018 |
| Data acquisition | NI USB 9162 | CTH2 | May 2016 | - | May 2018 |
| Tapping Machine | B&K 3207 | 1250 L | - | - | - |
| Data acquisition 8 channel | NI | 1333L | April 2016 | - | April 2018 |
| Data acquisition and data analysis software Noise Lab Capture 4.0 | DELTA | - | - | - | - |
| Acoustical software Dirac | B&K | - | - | - | - |
| Acoustical software Reverb.08 | DELTA | - | - | - | - |

Table 2
Instruments used for the tests.

Graph Sheets 1-8

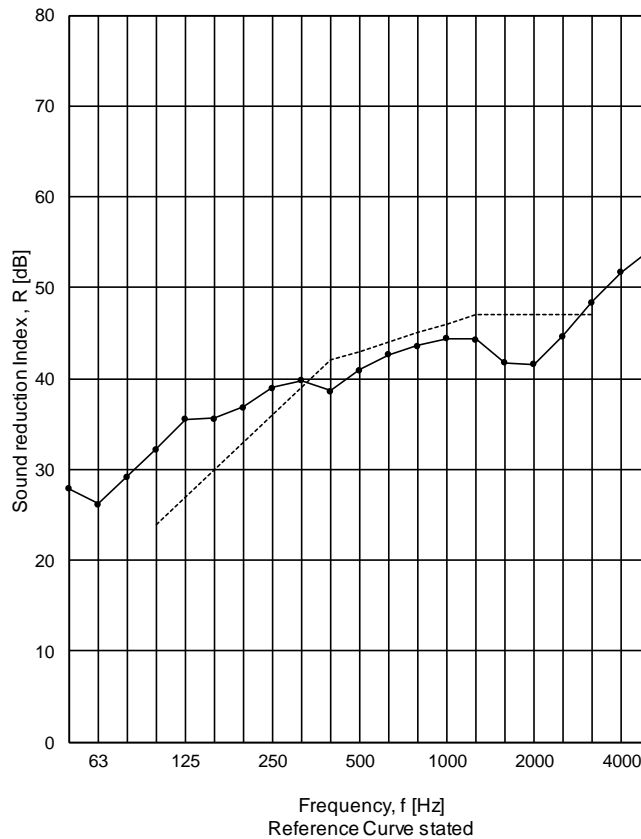


Laboratory measurement of sound reduction Index according to EN ISO 10140:2010

Customer: Sika Services AG
 Date of test: 26 march 2018
 Description of Test Specimen: SFM Visco EM 5
 Test specimen mounted by: The Client
 Place of measurement: Danish Technical University, Lyngby, Denmark

Test Areas, S: 10.0 m²
 Mass pr unit area: 23.8 kg/m²
 Temperature of air 004: 16.2 °C
 Humidity of Air 004: 42 % RH
 Temperature of air 904: 14.7 °C
 Humidity of Air 904: 47 % RH
 Source Room Volumen: 243 m³
 Receiving Room Volumen: 230 m³

| Frequency f [Hz] | R' 1/3-octave [dB] |
|------------------------|--------------------------|
| 50 | 27.9 |
| 63 | 26.2 |
| 80 | 29.2 |
| 100 | 32.3 |
| 125 | 35.5 |
| 160 | 35.6 |
| 200 | 36.8 |
| 250 | 39.0 |
| 315 | 39.8 |
| 400 | 38.6 |
| 500 | 40.9 |
| 630 | 42.6 |
| 800 | 43.6 |
| 1000 | 44.4 |
| 1250 | 44.3 |
| 1600 | 41.7 |
| 2000 | 41.5 |
| 2500 | 44.7 |
| 3150 | 48.4 |
| 4000 | 51.7 |
| 5000 | 54.1 |



Weighted sound reduction index according to EN ISO 717-1:2013:

$$R'_w = 43 \text{ dB}$$

Evaluation based on laboratory measurement results obtained by an engineering method EN ISO 10140: part 1 (2015), and part 2, 3 and 5 (2010)

DELTA, 17 april 2018

Leif Ødegaard, DELTA Acoustics

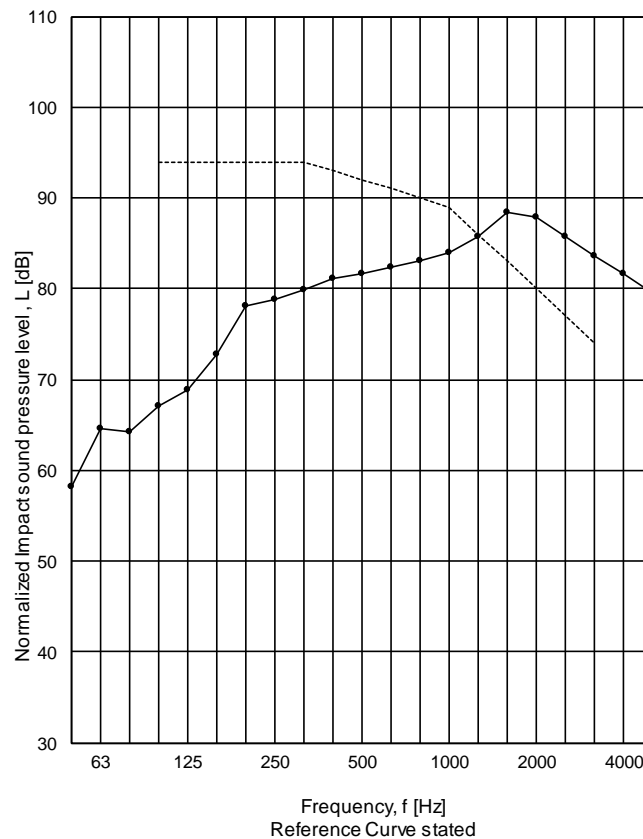


Laboratory measurement of normalized impact sound pressure level according to EN ISO 10140:2010

Customer: Sika Services AG
 Date of test: 26 march 2018
 Description of Test Specimen: SFM Visco EM 5
 Test specimen mounted by: The Client
 Place of measurement: Danish Technical University, Lyngby, Denmark

Test Areas, S: 10.0 m²
 Mass pr unit area: 23.8 kg/m²
 Temperature of air 004: 16.2 °C
 Humidity of Air 004: 42 % RH
 Temperature of air 904: 14.7 °C
 Humidity of Air 904: 46.6 % RH
 Source Room Volumen: 243 m³
 Receiving Room Volumen: 230 m³

| Frequency f [Hz] | L 1/3-octave [dB] |
|------------------------|-------------------------|
| 50 | 58.2 |
| 63 | 64.6 |
| 80 | 64.2 |
| 100 | 67.1 |
| 125 | 68.8 |
| 160 | 72.7 |
| 200 | 78.0 |
| 250 | 78.9 |
| 315 | 79.8 |
| 400 | 81.1 |
| 500 | 81.7 |
| 630 | 82.3 |
| 800 | 83.1 |
| 1000 | 83.9 |
| 1250 | 85.8 |
| 1600 | 88.4 |
| 2000 | 87.9 |
| 2500 | 85.8 |
| 3150 | 83.6 |
| 4000 | 81.6 |
| 5000 | 79.6 |



Weighted normalized impact sound pressure level according to EN ISO 717-2:2013:

$L_{n,w} = 92$ dB Calculated Impact Insulation Class IIC according to E989 ASTM : 18 dB

Evaluation based on laboratory measurement results obtained by an engineering method EN ISO 10140:part 1 (2015), and part 3, 4 and 5 (2010)

DELTA, 19 October 2018

Leif Ødegaard, DELTA Acoustics

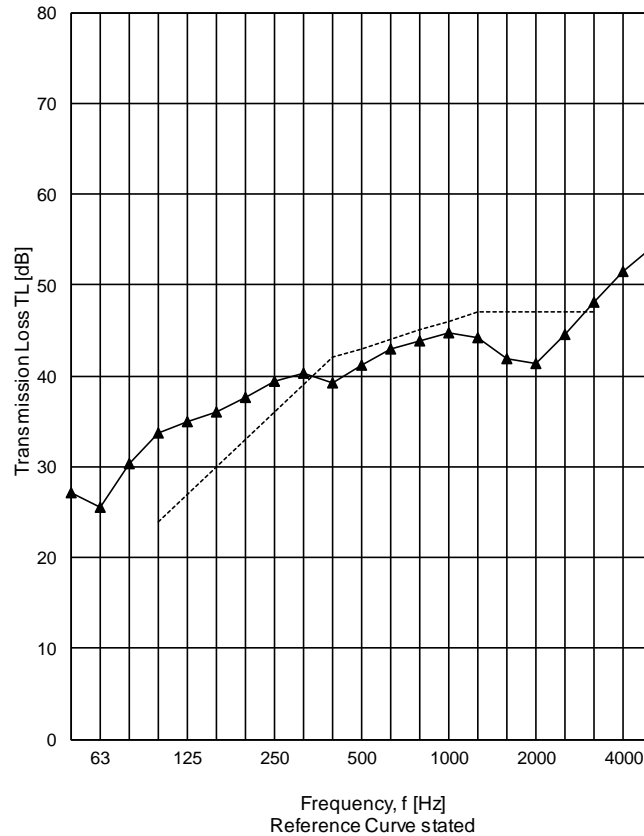


Laboratory measurement of airborne sound transmission loss according to ASTM E2963-16

Customer: Sika Services AG
 Date of test: 26 march 2018
 Description of Test Specimen: SFM Visco EM 5
 Test specimen mounted by: The Client
 Place of measurement: Danish Technical University , Lyngby, Denmark

Test Areas, S: 10.0 m²
 Mass pr unit area: 23.8 kg/m²
 Temperature of air 004: 16.2 °C
 Humidity of Air 004: 42 % RH
 Temperature of air 904: 14.7 °C
 Humidity of Air 904: 47 % RH
 Source Room Volumen: 243 m³
 Receiving Room Volumen: 230 m³

| Frequency f [Hz] | TL 1/3-octave [dB] |
|------------------------|--------------------------|
| 50 | 27.1 |
| 63 | 25.6 |
| 80 | 30.3 |
| 100 | 33.6 |
| 125 | 34.9 |
| 160 | 35.9 |
| 200 | 37.7 |
| 250 | 39.4 |
| 315 | 40.3 |
| 400 | 39.2 |
| 500 | 41.2 |
| 630 | 42.8 |
| 800 | 43.9 |
| 1000 | 44.7 |
| 1250 | 44.3 |
| 1600 | 41.8 |
| 2000 | 41.3 |
| 2500 | 44.5 |
| 3150 | 48.1 |
| 4000 | 51.4 |
| 5000 | 54.2 |



Weighted Sound Transmission Class according to ASTM E413 - 10 Classification for Rating Sound Insulation

STC = 43 dB

Evaluation based on laboratory measurement results obtained by an engineering method

DELTA, 17 april 2018

Leif Ødegaard, DELTA Acoustics

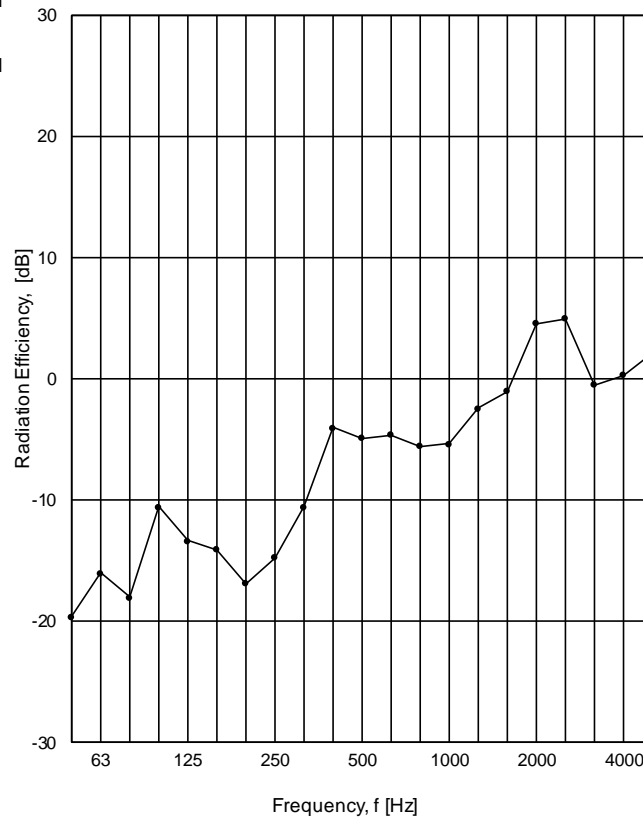


Laboratory measurement of radiation efficiency according to ASTM E2963-16


Customer: Sika Services AG
 Date of test: 26 march 2018
 Description of Test Specimen: SFM Visco EM 5
 Test specimen mounted by: The Client
 Place of measurement: Danish Technical University, Lyngby, Denmark

Test Areas, S: 10.0 m²
 Mass pr unit area: 23.8 kg/m²
 Temperature of air 004: 16.2 °C
 Humidity of Air 004: 42 % RH
 Temperature of air 904: 14.7 °C
 Humidity of Air 904: 47 % RH
 Source Room Volumen: 243 m³
 Receiving Room Volumen: 230 m³

| Frequency f [Hz] | L _σ 1/3-octave [dB] |
|------------------------|--------------------------------------|
| 50 | -19.7 |
| 63 | -16.1 |
| 80 | -18.1 |
| 100 | -10.6 |
| 125 | -13.5 |
| 160 | -14.2 |
| 200 | -17.0 |
| 250 | -14.8 |
| 315 | -10.6 |
| 400 | -4.1 |
| 500 | -5.0 |
| 630 | -4.7 |
| 800 | -5.6 |
| 1000 | -5.4 |
| 1250 | -2.5 |
| 1600 | -1.1 |
| 2000 | 4.5 |
| 2500 | 4.9 |
| 3150 | -0.6 |
| 4000 | 0.2 |
| 5000 | 2.1 |



Evaluation based on laboratory measurement results obtained by an engineering method

DELTA, 17 april 2018

 Leif Ødegaard, DELTA Acoustics

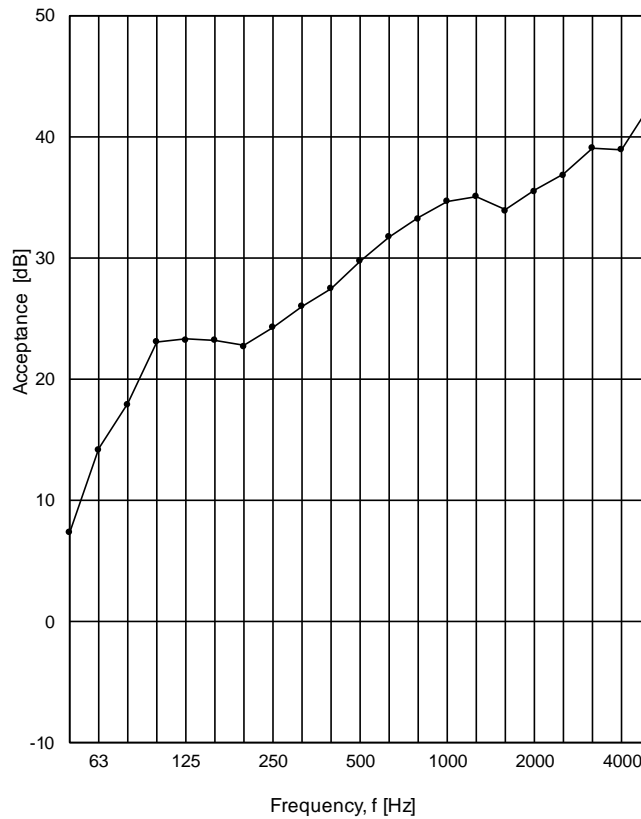


Laboratory measurement of acceptance according to ASTM E2963-16

Customer: Sika Services AG
 Date of test: 26 march 2018
 Description of Test Specimen: SFM Visco EM 5
 Test specimen mounted by: The Client
 Place of measurement: Danish Technical University, Lyngby, Denmark

Test Areas, S: 10.0 m²
 Mass pr unit area: 23.8 kg/m²
 Temperature of air 004: 16.2 °C
 Humidity of Air 004: 42 % RH
 Temperature of air 904: 14.7 °C
 Humidity of Air 904: 47 % RH
 Source Room Volumen: 243 m³
 Receiving Room Volumen: 230 m³

| Frequency f [Hz] | L _A 1/3-octave [dB] |
|------------------------|--------------------------------------|
| 50 | 7.3 |
| 63 | 14.2 |
| 80 | 17.9 |
| 100 | 23.0 |
| 125 | 23.2 |
| 160 | 23.2 |
| 200 | 22.7 |
| 250 | 24.2 |
| 315 | 26.0 |
| 400 | 27.4 |
| 500 | 29.7 |
| 630 | 31.7 |
| 800 | 33.2 |
| 1000 | 34.7 |
| 1250 | 35.0 |
| 1600 | 33.9 |
| 2000 | 35.5 |
| 2500 | 36.8 |
| 3150 | 39.0 |
| 4000 | 38.9 |
| 5000 | 42.6 |



Evaluation based on laboratory measurement results obtained by an engineering method

DELTA, 17 april 2018

 Leif Ødegaard, DELTA Acoustics

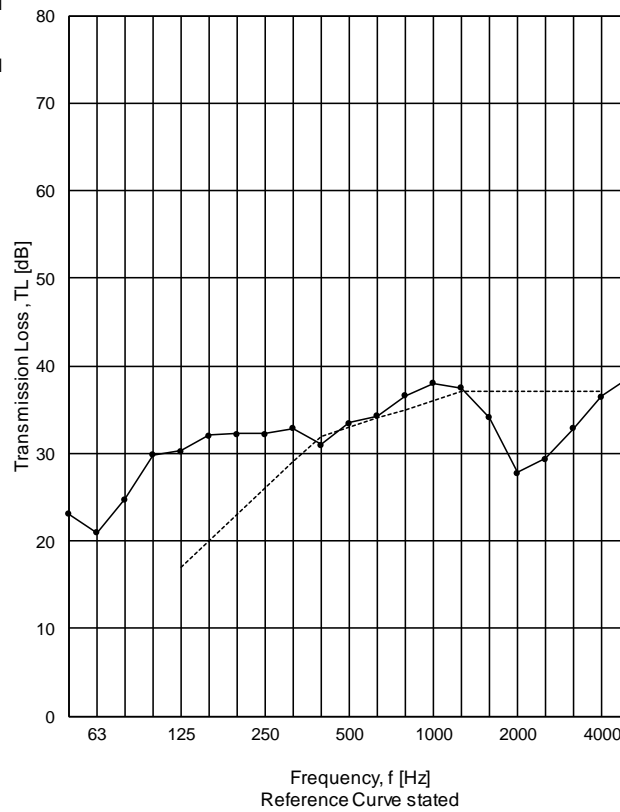


Laboratory measurement of airborne sound transmission loss according to ASTM E2963-16

Customer: Sika Services AG
 Date of test: 19 February 2018
 Description of Test Specimen: 6 mm Reference Deck
 Test specimen mounted by: The Client
 Place of measurement: Danish Technical University, Lyngby, Denmark

Test Areas, S: 10.0 m²
 Mass pr unit area: 45.0 kg/m²
 Temperature of air 004: 16.1 °C
 Humidity of Air 004: 42 % RH
 Temperature of air 904: 14.8 °C
 Humidity of Air 904: 43 % RH
 Source Room Volumen: 243 m³
 Receiving Room Volumen: 230 m³

| Frequency f [Hz] | TL 1/3-octave [dB] |
|------------------------|--------------------------|
| 50 | 23.1 |
| 63 | 20.9 |
| 80 | 24.7 |
| 100 | 29.8 |
| 125 | 30.3 |
| 160 | 32.0 |
| 200 | 32.2 |
| 250 | 32.2 |
| 315 | 32.8 |
| 400 | 31.0 |
| 500 | 33.4 |
| 630 | 34.3 |
| 800 | 36.6 |
| 1000 | 38.0 |
| 1250 | 37.4 |
| 1600 | 34.1 |
| 2000 | 27.8 |
| 2500 | 29.4 |
| 3150 | 32.9 |
| 4000 | 36.5 |
| 5000 | 38.7 |



Weighted Sound Transmission Class according to ASTM E413 - 10 Classification for Rating Sound Insulation

STC = 33 dB

Evaluation based on laboratory measurement results obtained by an engineering method

DELTA, 7 april 2018

Leif Ødegaard, DELTA Acoustics

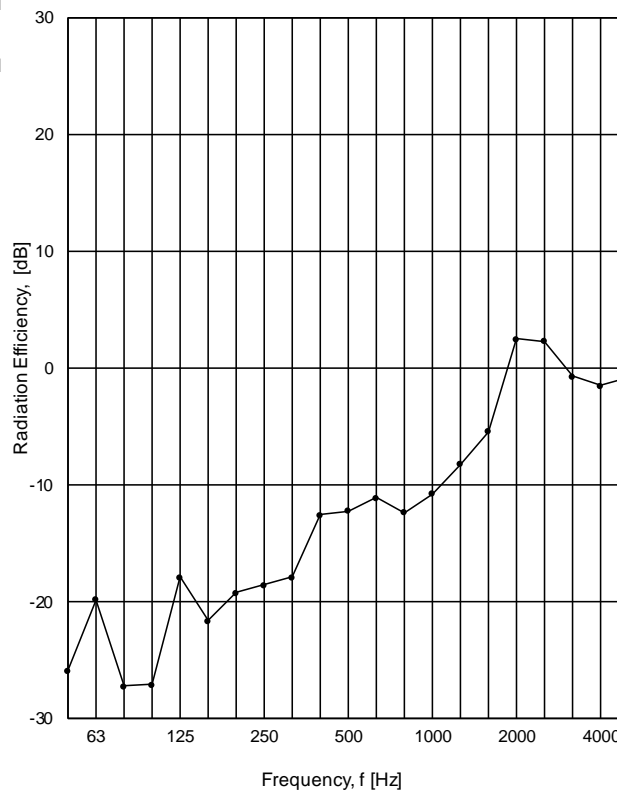


Laboratory measurement of radiation efficiency according to ASTM E2963-16


Customer: Sika Services AG
 Date of test: 19 February 2018
 Description of Test Specimen: 6 mm Reference Deck
 Test specimen mounted by: The Client
 Place of measurement: Danish Technical University, Lyngby, Denmark

Test Areas, S: 10.0 m²
 Mass pr unit area: 45.0 kg/m²
 Temperature of air 004: 16.1 °C
 Humidity of Air 004: 42 % RH
 Temperature of air 904: 14.8 °C
 Humidity of Air 904: 43 % RH
 Source Room Volumen: 243 m³
 Receiving Room Volumen: 230 m³

| Frequency f [Hz] | L _σ 1/3-octave [dB] |
|------------------------|--------------------------------------|
| 50 | -26.0 |
| 63 | -19.9 |
| 80 | -27.3 |
| 100 | -27.1 |
| 125 | -17.9 |
| 160 | -21.7 |
| 200 | -19.3 |
| 250 | -18.7 |
| 315 | -18.0 |
| 400 | -12.7 |
| 500 | -12.3 |
| 630 | -11.1 |
| 800 | -12.5 |
| 1000 | -10.8 |
| 1250 | -8.3 |
| 1600 | -5.5 |
| 2000 | 2.4 |
| 2500 | 2.2 |
| 3150 | -0.8 |
| 4000 | -1.6 |
| 5000 | -0.9 |



Evaluation based on laboratory measurement results obtained by an engineering method

DELTA, 7 april 2018

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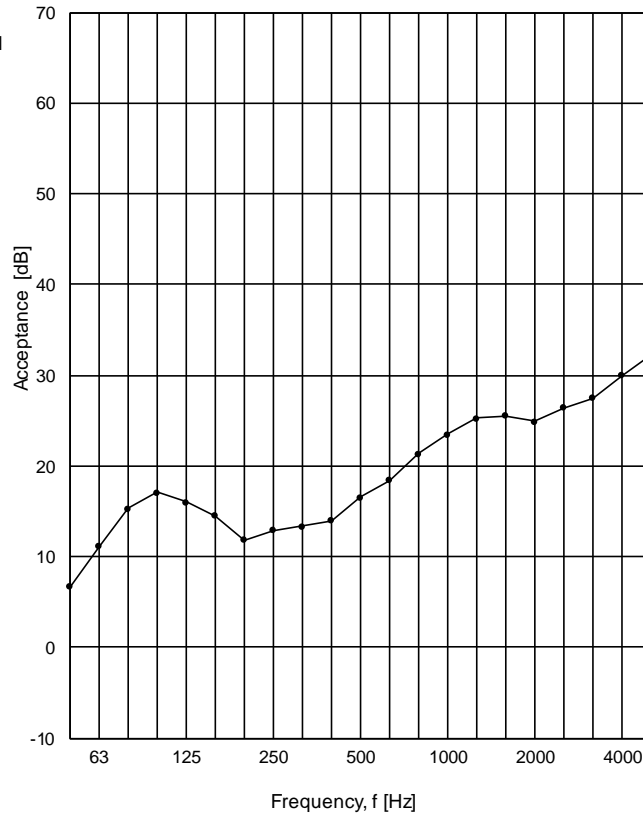


Laboratory measurement of acceptance according to ASTM E2963-16


Customer: Sika Services AG
 Date of test: 19 February 2018
 Description of Test Specimen: 6 mm Reference Deck
 Test specimen mounted by: The Client
 Place of measurement: Danish Technical University, Lyngby, Denmark

Test Areas, S: 10.0 m²
 Mass pr unit area: 45.0 kg/m²
 Temperature of air 004: 16.1 °C
 Humidity of Air 004: 42 % RH
 Temperature of air 904: 14.8 °C
 Humidity of Air 904: 43 % RH
 Source Room Volumen: 243 m³
 Receiving Room Volumen: 230 m³

| Frequency f [Hz] | L _A 1/3-octave [dB] |
|------------------------|--------------------------------------|
| 50 | 6.7 |
| 63 | 11.1 |
| 80 | 15.3 |
| 100 | 17.1 |
| 125 | 16.0 |
| 160 | 14.5 |
| 200 | 11.8 |
| 250 | 12.9 |
| 315 | 13.3 |
| 400 | 13.9 |
| 500 | 16.6 |
| 630 | 18.4 |
| 800 | 21.3 |
| 1000 | 23.5 |
| 1250 | 25.3 |
| 1600 | 25.5 |
| 2000 | 24.8 |
| 2500 | 26.4 |
| 3150 | 27.5 |
| 4000 | 29.9 |
| 5000 | 32.2 |



Evaluation based on laboratory measurement results obtained by an engineering method

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